# **INSTITUTE OF PLASMA PHYSICS OF THE CZECH ACADEMY OF SCIENCES**

# STRUCTURAL MODEL OF HIGH-TEMPERATURE VACUUM-SEALING FLANGE

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- COMPASS-U Tokamak
- Vacuum vessel, resilient metal seals
- Model definition
- Results

# TOKAMAK COMPASS UPGRADE



- TOKAMAKs are scientific devices to explore nuclear fusion
  - fusion of hydrogen isotopes (D+D, D+T)
  - hopefully future energy source of mankind!
- At IPP, we build a new tokamak called **COMPASS-Upgrade** 
  - Expected time of first run ~ 2026
  - Most important parameters
    - Toroidal mag. field 5 T
    - Plasma current
    - Major radius
    - Minor radius

| 5 T         | 2.1 T   |
|-------------|---------|
| 2 MA        | 0.35 MA |
| R = 0.894 m | 0.56 m  |
| a = 0.27 m  | 0.23 m  |
|             |         |

Old COMPASS



Fig. 1.: <sup>3</sup>/<sub>4</sub> Section of tokamak COMPASS-U. Position of plasma is indicated by **pink elipse**.



# COMPASS-U VACUUM VESSEL

- COMPASS-U vacuum vessel has a lot of (non-circular) ports
- Moreover, seals are required
  - To seal ultra-high vacuum (leak rate < 10-9 Pa.m3/s)
  - Withstand high temperature (500 °C)
- Only feasible option is to use resilient metal seals!



Fig. 2.: Illustration of resilient metal seals working diagram, illustration of seal cross-section © Gesellschaft für Dichtungstechnik mbH



Fig. 3.: Render of COMPASS-U vacuum vessel, ports visible



#### METAL SEALED FLANGES

- What are the challenges?
  - Huge compression force required! (~350 N/mm)
  - Only small springback allowed (~0.05 mm)
  - Flanges are **inverted** (bolts inside, seal outside) and not blind
  - Different materials all-around

Bolts and flanges are on the edge of possible, therefore the design optimization and verification is crucial!



Fig. 4.: Different resilient metal seals (Parker Hannifin)



#### **FLANGE MODEL**



# **BOLT BEAM MODEL**



- Bolts are modelled as a beams
- Default multiphysics beam-solid coupling doesn't support thermal expansion.
- Fortunately, COMSOL includes **Rigid connector**, which has much more settings!



Fig. 6.: Solid-beam connection node with resulting non-physical stress



Fig. 7.: Rigid connector node with thermal expansion subnode set up



Fig. 8.: Used beam element in the bolt hole



# SEAL NONLINEAR MODEL

- Main working mechanism of the resilient metal seal is the plastic deformation.
- We don't have the Nonlinear Structural Material Module!
- Just use the "External Materials" node!
  - Available in basic Solid mechanics module
  - Bit harder to use (you need to compile you own DLL)
  - But! COMSOL has a nice tutorials! [1] [2]

| Settings -<br>External Material                         |  |                                      |                  |   |       |
|---|--|--------------------------------------|------------------|---|-------|
| Label: External plastic                                 | • N  | Aaterial Contents                    |                  |   |       |
| Name: extmat1   | **   | Property                             | Variable         | Value                                   | Unit  |
| <ul> <li>Library for Windows, 64-bit</li> </ul>         |  | Density<br>Metarial model are set of | rho<br>(ass1 as  | 8250                                    | kg/m³ |
| Discard   |  | Extra library function string argume | {pari, p<br>args | {p_emoa, p_poisson, p_sigysu, p_e iiso} | Pa    |
| Library imported into model                             | Fig. 10.: External material parameters setup |                                      |                  |   |       |
| Filename: comsol_plastic.dll (Aug 23, 2022, 5:29:53 PM) |  |                                      |                  |   |       |
| Fig. 9 · External material node setup                   |  |                                      |                  |   |       |



[2] https://www.comsol.com/blogs/how-to-implement-elastoplasticity-in-a-model-using-external-materials/



Fig. 11.: Seal testing submodel (to verify external material)



Stress (MPa)

Fig. 12.: Resulting stress-strain relation from the submodel.



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# **MODEL RESULTS**



Fig. 13.: Resulting stress distribution in the flange and port, @300 K





- How to read the seal deformation? •
  - Both sides move, we need to subtract. ٠
- Linear extrusion is here to help! •
  - Part of the nonlocal couplings family •
  - It maps entities onto each other ٠
- After setup, you can just do •
  - deff = w linext1(w)•

Linear Extrusion 1 (linext 1) 

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Fig. 15.: Seal compression along the port, @300 K

#### **MODEL RESULTS**



## CONCLUSION

- We have optimized and verified the high-temperature flange design
  - Required bolt preload ~15 kN
  - Maximal springback reached < 0.05 mm
  - Bolt stress < 600 MPa



Fig. 16.: Tokamak COMPASS-U render